

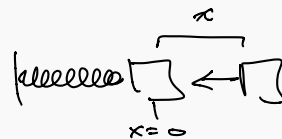
MAT303: Calc IV with applications

Lecture 15 - March 28 2021

Recap:

- We are interpreting $my'' + cy' + ky = f(t)$ as a mass-spring system.
- When $f(t) = 0$, saw that there are 3 regimes, depending on whether $c < 4km$.
- Last time, we saw how to solve the equation when $f(t)$ is nonzero.

mass
 ↓
 damping coef.
 ↓
 spring constant
 ↓
 ext force



$$F = kx$$

k brg: stiff spring

Today:

- Physical interpretation of $my'' + cy' + ky = f(t)$ when $f(t)$ is nonzero (Ch 3.6)
 - Resonance for damped and undamped forced oscillations
 - Transient and steady periodic solutions

Recall: (Lecture 13)

$$mx'' + kx = 0$$

- m : mass
- k : The constant such that Force = $k \cdot$ displacement

describes unforced, undamped oscillations.

$$= A \cos(\omega_0 t) + B \sin(\omega_0 t).$$

Solution is: $x = C \cos(\omega_0 t - \alpha)$, where

- $\omega_0 = \sqrt{\frac{k}{m}}$ does not depend on initial conditions.
- C and α depend on initial conditions

Examples:

- Mass on spring
- Guitar/piano string
- Bridge swinging side to side/up down
- Child on swing (pendulum)
- Wine glass



Now assume we put in an external force of F_0 :

$$mx'' + kx = F_0$$

Particular Solution:

- Trial solution $x(t) = A$

$$\begin{aligned} f(t) &= F_0 \\ f' &= 0 \\ f'' &= 0 \\ f''' &= 0 \end{aligned}$$

$$A \cdot F_0 \approx A$$

$$m x'' + kx = F_0 \quad F_0$$

$$x = A$$

$$\Rightarrow 0 + kA = F_0$$

$$\Rightarrow A = \frac{F_0}{k}$$

- Solution is $x(t) = F_0/k$

Full soln is

$$x = C \cos(\omega_0 t - \alpha) + \frac{F_0}{k}$$



Recall: (Lecture 13)

$$mx'' + kx = 0$$

- m : mass
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describes unforced, undamped oscillations.

Solution is: $x = C \cos(\omega_0 t - \alpha)$, where

- $\omega_0 = \sqrt{\frac{k}{m}}$
- C and α depend on initial conditions

Homog solns:

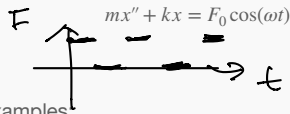
$$x = \sin(\omega_0 t)$$

$$x = \cos(\omega_0 t)$$

Examples:

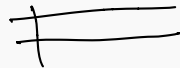
- Mass on spring
- Guitar/piano string
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- Wine glass

Now assume external force is periodic:



Examples:

- Mass on spring
- Guitar/piano string
- Bridge swinging side to side/up down
- Child on swing (pendulum)
- Wine glass



- F_0 : external force amplitude
- ω : (angular) frequency of ext. force

External force:

- Motor?
- Vibrations caused by other sounds
- Soldiers marching, wind blowing
- Adult pushing the swing
- Vibration caused by someone singing

Particular Solution:

$$f(t) = F_0 \cos(\omega t)$$

$$f'(t) = -\omega F_0 \sin(\omega t)$$

$$\text{Trial solution: } f''(t) = -\omega^2 F_0 \cos(\omega t)$$

$$x = A \cos(\omega t) + B \sin(\omega t)$$

$$x'' = -A\omega^2 \cos(\omega t) - B\omega^2 \sin(\omega t)$$

$$\text{So } -A\omega^2 \cos(\omega t) + \cancel{B\omega^2 \sin(\omega t)} = F_0 \cos(\omega t)$$

$$\text{So } B = 0, \quad A = \frac{F_0}{k - m\omega^2}$$

<https://www.desmos.com/calculator/tpirvpcwbe>

$$\text{So } x_p(t) = \frac{F_0}{k - m\omega^2} \cos(\omega t)$$

if $\omega = \omega_0$
 Total solution
 is
 $A t \cos(\omega t)$
 $+ B t \sin(\omega t)$

Now assume external force is periodic:

$$mx'' + kx = F_0 \cos(\omega t)$$

- m : mass
- k : The constant such that Force = $k \cdot$ displacement
- F : external force amplitude
- ω : (angular) frequency of ext. force

Examples:

- Mass on spring
- Guitar/piano string
- Bridge swinging side to side/up down
- Child on swing (pendulum)
- Wine glass

External force:

- Motor?
- Vibrations caused by other sounds
- Soldiers marching, wind blowing
- Adult pushing the swing
- Vibration caused by someone singing

$$x_p(t) = \frac{F_0}{k - m\omega^2} \cos(\omega t)$$

$$= \frac{F_0/m}{\frac{k}{m} - \omega^2} \cos(\omega t)$$

Solution:

If $\omega \neq \omega_0$: $x(t) = C \cos(\omega_0 t - \alpha) + \frac{F_0/m}{\omega_0^2 - \omega^2} \cos \omega t$

Now $\omega_0 = \sqrt{\frac{k}{m}}$

If $\omega = \omega_0$: $x(t) = C \cos(\omega_0 t - \alpha) + \frac{F_0}{2m\omega_0} t \sin \omega_0 t$

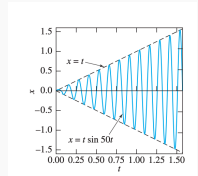


FIGURE 3.6.4. The phenomenon of resonance.

Observations

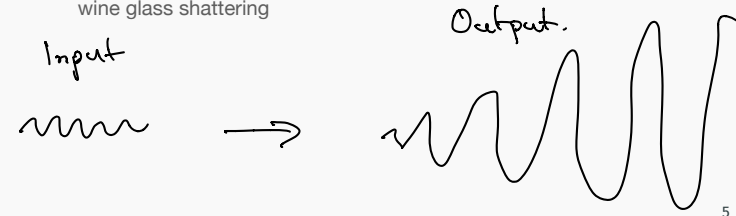
- New amplitude of x_p is much bigger than what if we just use $f(t) = F_0$

• Amplitude is $\frac{F_0/k}{1 - (\omega/\omega_0)^2}$

$$\frac{F_0/m}{\frac{k}{m} - \omega^2} \cdot \frac{\frac{m}{k}}{\frac{m}{k}} = \frac{F_0/k}{1 - \frac{\omega^2}{\omega_0^2}}$$

- Call $\rho = \frac{1}{|1 - (\omega/\omega_0)^2|}$ the amplification factor.

- As $\omega \rightarrow \omega_0$, this amplification goes to ∞ .
- This is the phenomenon of **resonance**.
- Roughly speaking: when external force is synchronized with natural frequency, amplitude gets very large.
- Causes bridge collapse, sympathetic resonance (music), wine glass shattering



No external force:

Damped forced oscillations

$$m\ddot{x} + c\dot{x} + kx = 0$$

- c : damping coefficient

Solution:

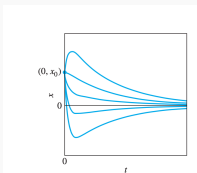


FIGURE 3.4.7. Overdamped motion: $x(t) = c_1 e^{r_1 t} + c_2 e^{r_2 t}$ with $r_1 < 0$ and $r_2 < 0$. Solution curves are graphed with the same initial position x_0 and different initial velocities.

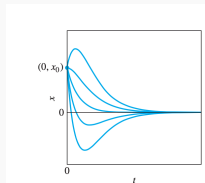


FIGURE 3.4.8. Critically damped motion: $x(t) = (c_1 + c_2 t)e^{-pt}$ with $p > 0$. Solution curves are graphed with the same initial position x_0 and different initial velocities.

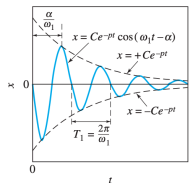


FIGURE 3.4.9. Underdamped oscillations: $x(t) = Ce^{-pt} \cos(\omega_1 t - \alpha)$.

As long as there is damping, $c > 0$, the solutions go to 0.

The solutions are called **transient**.

$$m\ddot{x} + c\dot{x} + kx = F_0 \cos(\omega t)$$

- F_0 : external force amplitude
- ω : (angular) frequency of ext. force

Trial solution: $x(t) = A \cos(\omega t) + B \sin(\omega t)$ or $x(t) = C \cos(\omega t - \alpha)$

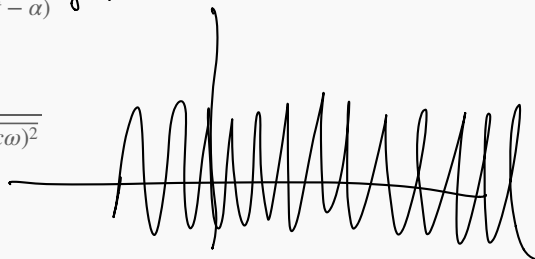
sp means steady period:

$$\text{Solution: } x_{sp}(t) = C \cos(\omega t - \alpha)$$

Where:

$$C = \frac{F_0}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}}$$

$$\tan(\alpha) = \frac{c\omega}{k - m\omega^2}$$



Observations:

- Amplitude is always finite, unlike undamped case
 - Amplitude close to F_0/k if ω is very small $\omega = 0 \Rightarrow C = \frac{F_0}{k}$.
 - Amplitude small if ω is very large $\omega \rightarrow \infty, C \rightarrow 0$.
 - Amplitude attains a maximum for some ω (minimize the denominator)

(Practical Resonance)

Full solution: $x = x_{tr} + x_{sp}$ In the long run, $x_{tr} \rightarrow 0$
 $x \approx x_{sp}$.

<https://www.desmos.com/calculator/o9js9ofznz>

initial conditions are forgotten.

Consider

$$mx'' + cy + kx = f(t)$$

- $m = 1, c = 2, k = 26$
- External force $f(t) = 82 \cos(4t)$
- $x(0) = 6, \quad x'(0) = 0$

Questions:

- Transient motion?
- Steady periodic oscillations?
- Practical resonance?

Homogeneous solution:

$$x'' + 2x' + 26x = 0$$

$$\Rightarrow r^2 + 2r + 26 = 0 \quad (\text{char eqn})$$

$$r = -1 \pm \sqrt{1 - 26} = -1 \pm 5i$$

So

$$x(t) = A e^{-t} \cos(5t) + B e^{-t} \sin(5t)$$

$$82 \cos 4t$$

Particular solution: (steady periodic soln).

$$\text{Trial Soln: } x = 2A \cos 4t + 2B \sin 4t$$

$$2x' = -8A \sin 4t + 8B \cos 4t$$

$$x'' = -16A \cos 4t - 16B \sin 4t$$

$$10A + 8B = 82 \Rightarrow A = 5$$

$$-8A + 10B = 0 \Rightarrow B = 4$$

$$\text{So } x_p = 5 \cos 4t + 4 \sin 4t$$

Full Soln:

~~Practical resonance?~~

$$x(t) = A e^{-t} \cos(5t) + B e^{-t} \sin(5t) + 5 \cos 4t + 4 \sin 4t$$

Note: as $t \rightarrow \infty$, $x \approx x_{sp}$.
regardless of initial conditions.

LCs:

$$A = 1, \quad B = -3$$

Practical Resonance:

$$C = \frac{F_0}{\sqrt{(26 - \omega^2)^2 + (26\omega)^2}}$$

amplitude: Maximize C by

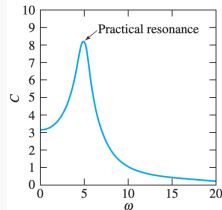


FIGURE 3.6.9. Plot of amplitude C versus external frequency ω .
minimizing $(26 - \omega^2)^2 + 26\omega^2$
at $\omega = \sqrt{26}$.

Today:

- Physical interpretation of $my'' + cy' + ky = f(t)$ when $f(t)$ is nonzero (Ch 3.6)
 - Resonance for damped and undamped forced oscillations
 - Transient and steady periodic solutions

Midterm: up to and including
Ch 3-8.

Next time:

- Ch 3.8 Endpoint problems

$$y'' + p(x)y' + q(x)y = 0; \quad y(a) = 0, \quad y(b) = 0.$$

